

## IAQ 2020: Indoor Environmental Quality Performance Approaches

### Ventilation Effectiveness of Alternating Façade-integrated Ventilation Devices in a Dwelling

Sven Auerswald, M.Sc.

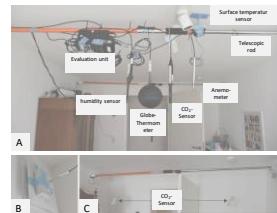
Fraunhofer Institute for Solar Energy Systems  
sven.auerswald@ise.fraunhofer.de | +49 (0) 761 / 45 88 - 5901



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## Outline/Agenda

- Push-pull devices
- Theory ventilation efficiency
- Ventilation levels standards vs. measurement site
- Results
- Conclusion



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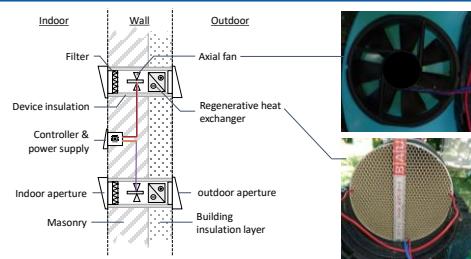
## Acknowledgements

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- Constanze Bongs
  - Head of Group Building Systems Technology Department Energy Efficient Buildings Fraunhofer ISE, Freiburg, Germany.

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## Push-pull devices = alternating façade-integrated ventilation devices

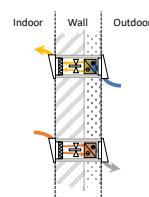
Device structure & installation



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## Learning Objectives

- Basics of push-pull devices = alternating façade-integrated ventilation devices
- Ventilation levels acc. to EU and German standards
- Push-pull devices provide mixing ventilation
  - Is this statement always valid?
  - If not: When is it valid?

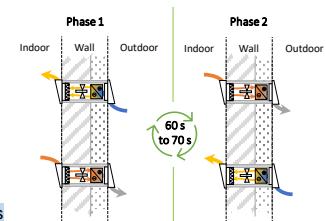


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## Push-pull devices

Operation principle & basic characteristics

- Alteration of flow direction to activate regenerative HX
- Pairwise decentralized installation in the façade
- T-ratio: ~85 %ΔT
- Max. V-flow: ~46 m<sup>3</sup>.h<sup>-1</sup>
- Particularly predestined for use in refurbishments



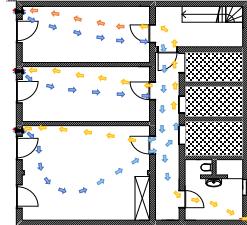
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## Push-pull devices

Ventilation concept

- Alteration of flow direction
- "Communication" of paired devices to:
  - Balance pressure & volume flow
  - Establish a flow pattern covering the whole indoor space
  - Not yet proven
- Ventilation characteristic: Mixed Ventilation
  - Possible but not always achieved



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## Theory: Ventilation Efficiency

Air Change Efficiency

$$\epsilon^a = \frac{\dot{m}}{2 \cdot \dot{m}_{nom}}$$

Prof. E. Skärst (1986):

"Rule no. 1 for designing effective ventilating systems is ... to design for air exchange efficiencies above 50%, in the occupied zone."

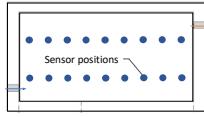


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## Theory: Ventilation Efficiency

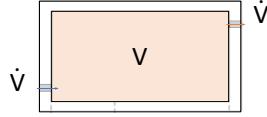
Air Change Efficiency

- average air age of a room  $\langle \bar{a}_j \rangle$   
= reciprocal of the provided air change rate inside the room (EN 16798-1)



$$\langle \bar{a}_j \rangle = \sum_{i=1}^N \bar{a}_i(x) \cdot N^{-1} = \frac{1}{\dot{m}}$$

- nominal time constant  $\tau_{nom}$   
= reciprocal of the nominal air change rate



$$\tau_{nom} = \frac{V}{\dot{V}}$$

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## Ventilation levels

standards vs. measurement site

• Input: 3 Adults, total area: 88.7 m<sup>2</sup>, ceiling height: 2.21 m

#	Ventilation level	DIN 1946-6 (2009) in m <sup>3</sup> ·h <sup>-1</sup>	DIN 1946-6 (2019) in m <sup>3</sup> ·h <sup>-1</sup>	EN 16798-1 (2021) in m <sup>3</sup> ·h <sup>-1</sup>	#	Pre-installed System (balanced + infiltration) in m <sup>3</sup> ·h <sup>-1</sup>	Pre-installed System (incl. exhaust only + infiltration) in m <sup>3</sup> ·h <sup>-1</sup>
2	Moisture protection	20.9	6.01		2		
3	Reduced ventilation	62.5	50.5		3		
4	Nominal ventilation	93.7	77.1	117	4	39.4 ± 8.6	96 ± 12
5	Intensive ventilation	125	104		5	48 ± 12	105 ± 17

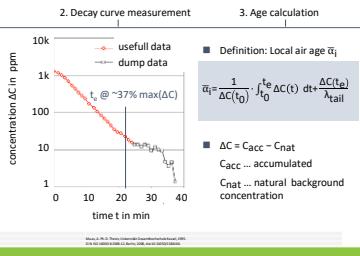
• German standards require lower ventilation rates than Europeans and have been further reduced.

• The system available for the study is an example for the assumption that existing decentralized residential ventilation systems tend to be too small.

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## Theory: Ventilation Efficiency

Air Change Efficiency



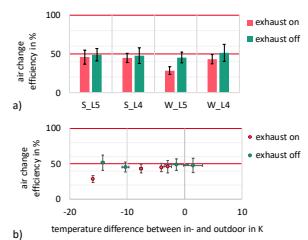
Author: S. L. Schäfer, Universität Regensburg, 2024, DOI: 10.5281/zenodo.1456163

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## Results

case	$\epsilon^a$ in %	$(0.1) \text{ h}^{-1}$
S_L5_e0	49.0 ± 0.4	0.21 ± 0.01
S_L4_e0	48 ± 10	0.24 ± 0.01
W_L5_e0	45.4 ± 7.0	0.27 ± 0.01
W_L4_e0	52 ± 11	0.26 ± 0.01
S_L5_e1	45.9 ± 9.0	0.47 ± 0.01
S_L4_e1	44.7 ± 6.0	0.1 ± 0.01
W_L5_e1	28.4 ± 5.0	0.28 ± 0.01
W_L4_e1	43.1 ± 6.0	0.38 ± 0.01



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## Conclusion

### Design phase:

- German standards require lower ventilation rates than Europeans and have been further reduced.
- The system available for the study is an example for the assumption that existing decentralized residential ventilation systems tend to be too small.

### Measurements:

- Very low air change rates in occupied zone due to low design volume flows
- Balanced cases slightly below mixed ventilation  $\epsilon^B = 0.48$
- Combined cases with exhaust only devices are short-circuiting  $\epsilon^B = 0.39$
- No clear impact of temperature gradients between in- and outdoors.
  - Possible reasons: mild climate conditions, good heat recovery performance or/and low ventilation flow does not penetrate the indoor space far enough

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Thank you!

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## Questions?

Sven Auerwald: [sven.auerwald@ise.fraunhofer.de](mailto:sven.auerwald@ise.fraunhofer.de)



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